

## SPECIFICATION

### GLASS CONTAINER FORMING MACHINE

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The present invention relates to I.S. (individual section) glass forming machines which form a parison at a blank station and then at a blow station, first blow the parison and then cool the blown parison to form a bottle and more particularly to the structure for blowing the parison and cooling the blown parison into a bottle and then cooling the bottle to a temperature below the annealing point so that the bottle can then be quickly cooled to room temperature.

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### BACKGROUND OF THE INVENTION

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The blowing operation is effected by a blow head. Conventionally the blow head is brought into position on top of (engaging) a blow mold at the blow station and provides air ("final blow") under pressure through a downwardly extending blow tube to the interior of the parison to blow the parison into contact with the interior of the blow mold. The parison could also be formed with vacuum or with a vacuum assist. The blown parison must then be formed into a bottle, i.e., cooled to the point where it is rigid enough to be gripped and removed from the blow station by a takeout mechanism. The outer surface of the blown parison is cooled by cooling the blow molds and the inner surface of the blown parison is cooled by the final blow air which continues to flow into the blown parison. U.S. Patent No. 4,726,833 discloses a state of the art blow head. Conventionally the cooling air escapes from the interior of the bottle through a permanently open exhaust. The size of the exhaust will be defined as a balance between inlet and outlet.

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Before a conventional takeout can be displaced from a remote location to a pick up location proximate the top of the formed bottle, the blow head, including the blow tube, must be displaced away from the blow mold. This

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displacement must be at least to a position where it will not interfere with an inwardly moving takeout. To speed up these steps, U.S. Patent No. 5,807,419, proposes a combined blow head and takeout mechanism. This mechanism  
5 permits the operation of takeout jaws as soon as the blow head, which engages the top of the blow molds during final blow, is slightly elevated, with the blow tube remaining fully extended and operating, following the formation of the bottle. The takeout jaws immediately  
10 reseal the blow head. The internal cooling of the bottle will accordingly continue as if the blow head was in place on top of the blow mold while the bottle is removed from the blow mold and carried to a dead plate on which it will be deposited. The cooling of the outer surface  
15 of the formed bottle stops with the opening of the blow molds.

U.S. Patent No. 4,508,557, discloses a dead plate arrangement for blowing cooling air around the bottle to provide additional outer surface cooling on the dead-plate.  
20 U.S. Patent No. 4,892,183 discloses a dual take-out mechanism which functions to alternately remove bottles from the blow station placing half on one output conveyor and the other half on a second output conveyor.

In all of these systems, the bottles once removed  
25 from the deadplate, will be conveyed into a Lehr which utilizes a series of burners to immediately reheat the bottles to a uniform higher temperature and then allows the bottles to cool slowly before being discharged from the Lehr.

30 Formed bottles have also been tempered in separate machinery by reheating the bottles and then simultaneously cooling the inner and outer glass surfaces (see for example, U.S. Patent No. 2,309,290).

OBJECT OF THE INVENTION

It is an object of this invention to provide an I.S.  
5 machine which more effectively removes heat from the  
blown parison/formed bottle.

Other objects and advantages of the present  
invention will become apparent from the following portion  
of this specification and from the accompanying drawings  
10 which illustrate in accordance with the mandate of the  
patent statutes a presently preferred embodiment  
incorporating the principles of the invention.

BRIEF DESCRIPTION OF THE DRAWINGS

15 Figure 1 shows a perspective view of a blow head  
mechanism made in accordance with the teachings of the  
present invention;

20 Figure 2 shows a diagrammatic cross sectional view  
of the blow head of the blow head mechanism shown in  
figure 1;

Figure 3 shows a time versus pressure curve for the  
operating blow head ;

25 Figure 4 shows a logic diagram for the operation of  
the blow head mechanism shown in Figure 1;

Figure 5 is an enlarged elevational view in section  
of the blow head at the exhaust position;

30 Figure 6 shows an elevational view in cross section  
of the blow head mechanism made in accordance with the  
teachings of the present invention;

Figure 7 shows a view of the cooling tube shown in  
figure 6 taken at 7-7 thereof;

Figure 8 is an elevational sectional view of the  
bottom of the cooling tube;

35 Figure 9 is a view taken at 9-9 of figure 8,

Figure 10 is a first displacement profile illustrating the vertical displacement of the cooling tube during the blowing and cooling of the parison to form a bottle;

Figure 11 is a second displacement profile illustrating the vertical displacement of the cooling tube during the blowing of the parison and the cooling of the parison to form a bottle; and

Figure 12 is a logic diagram illustrating the application of the displacement profile illustrated in Figures 10 and 11.

Figure 13 shows a perspective view of a takeout mechanism made in accordance with the teachings of the present invention;

Figure 14 shows an elevational view in section of the takeout mechanism shown in figure 13;

Figure 15 is a view taken at 15-15 of figure 14;

Figures 16-21 illustrate one of the pair of synchronized takeout/deadplate mechanisms of the present invention moving through a single cycle;

Figure 22 is a view taken at 22-22 of figure 21;  
and

Figures 23-28 illustrate the synchronism of a pair of takeout assemblies with their associated deadplate mechanisms;

Figure 29 is an oblique view of a the deadplate mechanism shown in figures 16-21;

Figure 30 is an oblique view of the mechanism for opening and closing the can doors;

Figure 31 is a logic diagram illustrating the operation of the temperature sensor; and

Figure 32 is a temperature vs. position curve for the formed bottle.

#### 35      BRIEF DESCRIPTION OF THE PREFERRED EMBODIMENT

Figure 1 shows a blow head mechanism 10 of the blow station of an I.S. machine. A triple gob machine is illustrated, and three blow molds 12 arranged side by side

are shown. A blow head arm 16 supports three blow heads 18. The blow head arm 16 is mounted on a vertical post 20 coupled to an electronic (servo) motor 22 which causes the blow head arm to move up and down. The post also rotates 5 via a scroll cam (not shown) defined in a housing 24. Such up and down and rotary movement of the post 20 causes the blow heads 18 to be displaced between a retracted "off" position and an advanced "on" position, as shown in Figure 1, at which the blow heads 18 engage the top of the 10 blow molds 12. The operation of the servomotor is controlled by a control C (26). Air under pressure is supplied from a suitable source 27 to a pressure regulator P (29) which will set the desired pressure for final blow as defined by the control C/26.

15 A blow head 18 is shown diagrammatically, in section, in Figure 2. The blow head 18 has an air inlet 34 leading to a blow tube 36 which extends downwardly into the parison 38. The blow head surrounds the finish 40 of the parison. Final blow air blows the parison and then 20 cools the interior surface of the blown parison. Air exhausts upwardly between the blow tube 36 and the parison into a chamber 41 and out through an adjustable exhaust 42. A pressure sensor 44 is arranged to monitor the air pressure in the chamber 41 (closely related to the 25 pressure of the air within the parison).

Figure 3 shows changes that have been discovered in the pressure  $P$  in the chamber 41 vs. time  $T$  plot. At about  $T_1$  seconds (.05 seconds, for example) after the time  $T_0$  when blowing pressure is applied through the air inlet 34, the pressure in the chamber begins to rise. The 30 pressure increases to an initial high  $P_1$  and drops to  $P_2$  (it is believed that this occurs as the parison rapidly expands). At  $T_2$  (.15 seconds, for example) the parison is blown against the blow mold and the pressure once again 35 increases until it reaches a steady state pressure  $P_3$  which continues until the blow head is removed more than

one second following the application of final blow. The pressure sensor 44 supplies data to the control C (26). While the curve has been discussed relative the blowing of the parison with pressure, it would be the same with 5 vacuum assist or with the blowing of the parison with negative pressure (i.e., vacuum).

The control first functions to Displace The Blow Head To The "On" Position And Set Blow Head Air Pressure To "Final Blow" Pressure 50. "Final blow" pressure can be 10 selectively set and is a pressure that will result in the parison being properly blown. Conventionally, "final blow" air is supplied from 20-30 PSI. Higher pressure will result in a defective bottle. Pressure is applied for a time T2 which is the time required to blow the 15 parison (until the query "Has Time T2 Passed" 52 is answered in the affirmative). The operator may empirically define and Input T2 54. Alternatively the control can "Determine T2" 56 by determining the location of the negative peak (a local minimum) at P2 (This 20 negative peak may be slightly delayed from the instant when the bottle is fully blown and a correction could then be applied). In practice T2 could be periodically determined with the control receiving updated T2 input. The operator may also reduce T2 if he finds that the 25 blowing of the parison will not be effected. With the parison blown, the control will "Raise The Blow Head "X" And Set Blow Head Air Pressure To "Internal Cooling" Pressure" 58 (X and "internal Cooling" Pressure can be selectively set). This second "on" position is the escape 30 position. The cooling flow is no longer limited by the size of the blow head exhaust. The volume of cooling flow for the remaining second or more before the blow head is turned "off" will be very substantially increased. "Internal cooling" air, can be supplied at a pressure 35 which is substantially higher than final blow air. For example internal cooling air can be supplied at 45 PSI

since this is a commonly available air supply. Internal cooling air will be supplied at a pressure sufficient to maintain at least a minimum desired pressure within the bottle. This cooling flow could continue until time T3  
5 (until the query "Has Time T3 Passed?" 60) is answered in the affirmative whereupon the control will "Displace The Blow Head To The "Off" Position" 62.

When the blow head is lifted the vertical distance "X" to the escape position (figure 5), exhaust air will be  
10 directed by the selectively concavely contoured annular recess surface 63 of the interior opening of the lower portion of the blow head to direct cooling air at the outer vertical surface of the finish.

Each blow head (figure 6) has a central axial hole 70 configured to matingly receive the blow tube 34. The blow tube is displaceable vertically but is restrained from rotating by a pair of guide keys 72 which engage opposed flats 74 (Fig.7) on the outer diameter of the blow tube.  
15 The top end portion 76 of the blow tube is cylindrical and threaded having an outer diameter larger than the spacing between opposed flats and the keys accordingly function as a down stop for the blow tube. A blow tube support and drive assembly 78 is mounted on the top surface 79 of the blow head arm with a number of screws 80. The assembly  
20 has an air manifold 82 including a link 84 communicating with a final blow air duct 85 in the arm, an overhead distribution manifold 86 and three air distribution legs 88 which depend vertically from the overhead distribution manifold.  
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30 Located within each distribution leg is the top portion 90 of a drive member 92 having a threaded internal diameter 94 extending downwardly through the top portion, through a driven gear portion 96 and then through a lower portion 98 which extends downwardly through the blow head  
35 mounting assembly 100. The O.D. of the drive member 92 is rotatably supported by three bearings 102. The internal

thread of the drive member I.D. threadedly receives the threaded top end portion 76 of the blow tube and vertical displacement of the blow tube will accordingly result whenever the driven gear portion 96 is rotated. Rotation 5 will be controlled by an electronic motor 104 coupled to a drive gear 106. The drive gear engages adjacent driven gear portions of the left two driven gear portions to drive the left two drive members 92 and an idler gear 108 between the right hand pair of driven gear portions 96 10 drives the right hand drive member.

The bottom of the blow tube 34 (figure 8) has an annular relief 110 defined in the I.D. The annular upper collar 111 (which is supported by an "X" frame 112) of an air deflector assembly 114, is press fit into the annular relief. Integral with and suspended from the frame 112 is a deflector 116 having an annular concave surface 118 that will divert a portion of the downwardly directed air stream radially outwardly towards the outer wall of the blown parison with the remainder flowing downwardly. 15 20 Figure 6 shows the blown parison which when cooled becomes a bottle 10 and shows the blow mold 12 which includes a bottom plate 11 and a pair of mold halves 12a, 12b.

Figure 10 illustrates an illustrative displacement profile for the blow tube which will blow and cool the parison. The blow head is displaced to the "on" position with the blow tube at the "up" position (T1). The blow tube is then rapidly accelerated to a maximum velocity (V1) and held at that velocity until T2. The blow tube 25 is then decelerated to a lower velocity V2 at T3 and held at that velocity until T4 when it is decelerated to a stop at its "down" position (T5). The blow tube will then remain at the "down" position until T6. The blow tube 30 will then follow the same profile returning the blow tube to a stop at the "up" position. The blow head can then 35 be removed and the molds opened. The displacement

profile will be selected to achieve the desired cooling of the inner surface of the blown parison, i.e., the motion profile is configured to co-ordinate with the cooling requirements of the container. This co-ordination can be a co-ordination based on the heat times the mass of the bottle. As shown in Figure 6, the bottle has a long neck which has less glass to cool than the body of the bottle. And if the bottle was formed in a blow and blow process, the body of the bottle will be hotter relative to the long neck. As a result the velocity of the blow tube as it proceeds along the neck portion is co-ordinated with the heat pattern of the bottle (the amount of heat energy desired to be removed along the bottle) and is much faster traversing the long neck than is the velocity traversing the body. Accordingly more cooling will be directed to the body where it is needed. Where the bottom of the formed parison is thicker, even more cooling will be required and the dwell (T6-T5) at the bottom will result in a lot of cooling air being directed at the bottom. Cooling air will continue to rise up along the body and neck to achieve additional cooling when the blow tube is at the bottom (this will also happen at any vertical position). Figure 11 illustrates a variant displacement profile where the blow tube makes three cycles while the parison is blown and cooled. This co-ordination could also be a function of the shape of the bottle. For example the bottle might have a bulge which would not be effectively cooled by cooling air flowing upwardly from a nozzle located below the bulge. In this situation like the above cooling of a thicker base the displacement of the cooling nozzle might be either stopped at this bulge to allow more cooling air to be directed into the bulge or slowed down as it displaced upwardly across the bulge to the same effect. The forming process will also be relevant to this co-ordination. Thickness of the glass as a function of

height may vary. In a blow and blow process the upper portion of a container will tend to be colder than the bottom portion and vice versa for a bottle formed in a press and blow process.

5       Figure 12 illustrates a logic diagram for controlling the displacement with different cycles during the time when a parison is blown and cooled. Here the operator inputs the number "N" of cycles desired. The control will Define Time Blow Head "Off" -Time Blow Head  
10      "On" 120, proceeds to Divide By "N" To Define Cycle Time 122 and then proceeds to Scale Blow Tube Displacement Profile For Cycle Time 124.

15      While the blown parison/formed bottle is in the blow mold external cooling will be effected by blowing cooling air through a series of circumferentially located cooling holes 19 defined in the blow molds which are supplied by an air plenum 21 to which the mold bottom plate 11 is secured.

20      A takeout mechanism is schematically illustrated in figure 13. Three bottles 10 which were formed in blow molds at the blow station are shown standing on the bottom plate 11 of an associated blow mold pair 12a,12b shown in the open position. The illustrated machine is a triple gob machine and accordingly three bottles 10 were  
25      formed. Once the molds are opened, a takeout assembly 140 of a takeout mechanism 150 grips the bottles. The takeout mechanism also includes a three axis support 160 for the takeout assembly that is suspended from a beam 170 that traverses the machine, i.e., spans the  
30      6,8,10,12,16, etc., individual sections of the machine. The three axis support, which includes an X axis drive 180, a Y axis drive 190 and Z axis drive 200, can take a great variety of forms including the form shown in U.S. Patent No. 4,892,183, which is incorporated by reference  
35      herein.

The takeout assembly has, at each bottle location, a blow tube 34 (figure 14). The blow tube support and drive assembly is the same as for the blow head mechanism except that the drive members 92 end at the driven gear  
5 portion and the guide keys 132 extend downwardly from the top wall 133 of the gripper housing 134 proximate the blow tube holes 135.

The takeout assembly also has a manifold housing 141 including an overhead distribution manifold 142 and three  
10 air distribution legs 143 which depend vertically from the overhead distribution manifold. Final blow (this includes air for final blow and/or internal cooling) depending on how the parison is being formed) air F.B./144 is supplied to the distribution manifold via a  
15 selectively controlled valve 145.

The base 164 of the manifold housing 141 is bolted onto the top wall 33 of the gripper housing 134 with a number of screws 165 with the driven gear portions 96, the drive gear 106 and the idler 108 located in a chamber  
20 located between the base of the manifold housing and the top wall of the gripper housing. The manifold housing has a pair of guide tubes 166 extending vertically upwardly from the top of the manifold housing which receive vertical guide rods 168 which are part of the Z  
25 axis drive 20.

As can be seen from figure 14, the gripper housing may start as a solid block. A through slot 171 having opposed horizontal keyways 172 is defined at each bottle location extending from the front of the gripper housing  
30 to the rear thereof. These slots receive front and back gripper brackets 174 (figure 15) each of which has integral vertical front 175 and horizontal bottom 176 panels extending completely across the gripper housing and vertical transverse (front to back) panels 178 which  
35 include horizontal keys 179 which are received by the keyways 172. The vertical front panels 175 are open 177

between the vertical transverse panels to allow easy flow of the air from the interior of the bottle to atmosphere. Secured within each of a pair of through holes 173 which extend through the gripper housing from the front to the  
5 back of the housing is a double acting cylinder 181 including opposed piston and rod assemblies 182. A pair of screws 183 connect each gripper bar to the piston rods 184 on the side of the gripper bar and compression springs 186, located between the piston and the cylinder  
10 housing will normally maintain the gripper bars at the closed position. A locating plate 187 is secured to the front panel of the gripper bracket with a rod receiving hole 188 to locate the axis of the rod. Air under pressure is supplied via a valve 191 from a source of  
15 gripper air G.A./192 to the center of each cylinder to open the gripper bars. The gripper bars may have selectively sized semicircular inserts (not shown) so that the closed gripper bars will grip the formed bottles on the finish of the bottles.

20 Figures 16-22 schematically illustrate how three bottles that have just been formed in the blow station of a triple gob I.S. machine standing ready for pick-up (with the blow molds withdrawn) are sequentially processed by a takeout assembly. The takeout assembly  
25 will remove bottles from the blow station and deposit them on a conveyor 15 and the bottles will then be conveyed into a cooling tunnel 17 (the tunnel will isolate the hot air from an operator who may have to enter the space between the conveyors to service either  
30 the conveyor or the machine). The takeout assembly 140 is shown in Figure 16 at the first deadplate position. Bottles have been formed in the blow molds 12. The molds open and the takeout assembly moves longitudinally to the pickup location shown in Figure 17 where the formed  
35 bottles will be gripped. The gripped bottles will be removed from the pickup position and carried back to the

first deadplate position (Figure 18). In the event that the bottles are to be rejected, the gripper jaws can be opened at the first deadplate position to drop the rejected bottles into a cullet removal chute 13. The gripped bottles, supported at the blow station, are supported next to doorways or openings in associated cooling cans 220 which are supported on a deadplate mechanism 240 which is at its park position. The deadplate mechanism now moves horizontally, transversely towards the gripped bottles to the first deadplate position (until the gripped bottles are supported centrally within their associated cooling can) and the doors of the cooling cans are then closed (this is shown with the circle of the cooling can being a closed circle) figure 19. The takeout assembly and the deadplate mechanism then conjointly horizontally transversely move to a conveyor location adjacent a first, right side conveyor 15 (Figure 20), the cooling can doors open and the takeout assembly then moves transversely away from the deadplate mechanism (figure 21) and then vertically downwardly from the up position to the down, deposit position (Figure 22) to place the gripped bottles on the conveyor whereupon they will be released. The takeout is then returned to the up position and the deadplate mechanism and the takeout assembly will then be conjointly transversely displaced back to their initial positions shown in Figure 16. Again the takeout can be displaced with sequential or simultaneous x and y movements. When molds are to be changed, both deadplate mechanisms can be displaced to the conveyor location to open up space for the operator.

With the bottles (Bottles No. 1) removed from the blow station (figure 18), an invert mechanism (not shown) will deliver formed parisons to the blow station and the blow molds will close. The parison will be blown and cooled to form a bottle (figure 19) and the molds will

open so that the sequentially formed bottles (Bottles No. 2) can be removed (figure 20) by a second takeout assembly. This forming process will be repeated with the next formed bottles (Bottles No. 3) being removed by the  
5 first takeout assembly. The synchronous movements of the first takeout assembly and its associated deadplate mechanism and the second takeout assembly and its associated deadplate mechanism are illustrated in Figures 23-28.

10 During the time when the first takeout assembly is at the first deadplate position (figure 23), is displaced to the pickup position (figure 24) to grip a bottle, returns with the gripped bottles to the first deadplate position (figure 25), and waits for the first deadplate  
15 mechanism to move to the first deadplate position to capture the bottles and close the cooling can doors (figure 26), the second takeout assembly and second deadplate mechanism are located at the conveyor location adjacent a second, left side conveyor with bottles formed in the previous cycle located within the cans with the can doors closed. Before the first takeout assembly and first deadplate mechanism are displaced conjointly to the conveyor location adjacent the first conveyor (figure  
20 27), the doors to the cans of the second deadplate mechanism open and the second takeout assembly is transversely displaced to displace the gripped bottles to a deposit location over the second conveyor whereupon the second takeout assembly is lowered from the up deposit location to a down deposit location to locate the gripped  
25 bottles proximate the second conveyor. The gripped bottles are released and the second takeout assembly is raised to the up deposit location. As the first takeout assembly and first deadplate mechanism are displaced from the first deadplate position to the conveyor position  
30 35 proximate the first conveyor, the second takeout assembly

and second deadplate mechanism are conjointly displaced to their start locations (figure 28) to start their cycle again removing the next bottles (Bottles No. 2) formed in the blow station.

5       The basic cycle now repeats with the roles reversed for the first takeout assembly/deadplate mechanisms and the second takeout assembly/deadplate mechanism with the first takeout assembly/deadplate assembly returning to their start locations to receive the next formed bottles  
10      (Bottles No. 3). While the displacement of the takeout arm from the conveyor location to the pick up location is shown with sequential X and Y movements it should be understood that such movements could occur simultaneously.

15      Figures 29 and 30 illustrate a deadplate mechanism which has a plenum chamber 194 which is supplied cooling air C.A./195 controlled by a selectively actuated valve V/196. Cooling air is available throughout the entire period during which a bottle is located within a can and for longer periods to cool the can either before or after a bottle is located within the can. Cooling air enters the cans 220 through holes 198 in the top surface 199 of the plenum chamber blowing up against the bottom of a bottle supported above the top surface by a takeout  
25     assembly and up the space between the suspended bottle and the inside wall 101 of the can, leaving the can through the can opening 103 at the top of the can. The plenum chamber is supported for Y-axis displacement by suitable rods 105 and is displaced by a Y-axis drive 107.  
30      Figure 21 schematically illustrates the door displacement mechanism for the deadplate mechanism cans. The doors 109 are coaxially mounted on a gear ( a worm gear for example) 206 which is supported for rotation about its axis. Operatively connected with each gear is a worm  
35     (for example) 208 which is displaced by a drive

209 having a motor 210 connected to the worm via a rotary  
to linear converter 212 (alternates such as rack and  
pinions may be used).

The interior surface of a can is configured so that  
5 cooling air admitted into the can through the bottom  
inlet holes 198 in the top surface of the plenum chamber  
will follow the surface of the bottle during its passage  
to the exit hole 103. Air flow to a can will occur as  
desired to achieve the cooling of the bottle but in the  
10 preferred embodiment air flow is continuous from the time  
a bottle enters a can to the time a bottle leaves the  
can.

A temperature sensor 125 secured to one or more of  
the cans provides temperature data which should be stable  
15 over time (data would be compared at the same point in  
the cycle). The control C/26 which receives this data  
determines whether "Sensed Temperature At Can T° +/- X°"  
126 (T and X can be manually or automatically inputted)  
and where the answer is answered in the negative, the  
20 control will "Reject The Bottles" 127. Where the cullet  
chute is located in the center, the deadplate mechanism  
can be displaced back to its ready position, the doors of  
the can can be opened, the takeout can be displaced to a  
position over the cullet chute and the bottles can be  
25 released.

The blow tube will be oscillated between the up and  
down positions with a displacement profile matched to the  
cooling requirements of the bottle from the moment the  
takeout assembly is lowered to its bottle gripping  
30 position until the gripped bottle is deposited onto the  
conveyor. As with the blowhead a convenient algorithm  
for defining this oscillation is shown in figure 12 and  
numerous cycles will occur while the bottle is gripped by  
a takeout assembly.

Referring to figure 32 which tracks the thermal energy of the object along the glass forming process, it can be seen that the thermal energy continuously decreases from the time the parison is blown in the blow  
5 mold to the time the bottle is discharged from the cooling tunnel. Thermal energy is first removed by the internal cooling of the blown parison within the blow mold and the conjoint cooling of the blown parison by the blow molds. Cooling then continues from the time a  
10 bottle is gripped by a takeout assembly to the time it is deposited on a conveyor and then cooling occurs as the bottle proceeds along the conveyor.

As can be seen from figure 31, the thermal energy of the bottle has been reduced to the point where the bottle  
15 is fully tempered before it is deposited on the conveyor and accordingly further cooling can accordingly take place at a rapid rate without causing defects in the container. Referring to figure 16, conveyor cooling which may be within a tunnel or not. Cooling would  
20 continue for a distance that would be much shorter than the length of a conventional Lehr, perhaps as short as about 25 feet. If it is within a tunnel, the tunnel may be divided up into a number of cooling zones each of  
25 which has a fan 300 which supplies shop air to an inlet 302 within the tunnel directing the air upstream. Upstream of the inlet is an exhaust 304 which discharges the cooling air from the tunnel. If there is no tunnel the fans will simply blow cooling air at the bottles. When the bottles are sufficiently cooled they will be  
30 discharged from the conveyor for further processing which could include inspection and packing or filling.